

# PERFORMANCE OPTIMIZATION OF EXISTING BOILERS AT SHAKARGANJ LIMITED, JHANG

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## ABSTRACT

There are numerous well-documented phenomena that plague the efficient operation of bagasse boilers. Key Parameters that influence boiler combustion and operation are studied with the aid of combustion and flue gas analyzer. Combustion stability and efficiency is linked to various parameters such as fuel moisture and air temperatures supplied to the boiler and are investigated in this paper as part of a case study. The paper highlights number of modern developments that have been implemented at Shakarganj Limited, Jhang to optimize the existing boiler design to enhance the Boiler capacity from 65 tph to 80 tph. Along with that impact of various devices on performance improvement of boiler has been shown with experimental data for comprehensive evaluation of boiler operation and combustion efficiency. Results of suitable measures after installation of low cost retrofits to reduce losses including combustion instabilities, unburnt fuel, moisture in fuel and deposition of ash on tubes are also part of the paper.

**Key Words:** Bagasse, Boiler, Air Heater, Economizer, Spreaders, Bagasse Dryer, Ash Cyclones, Flue Gases

## INTRODUCTION

Shakarganj Limited, incorporated in 1967, transform renew-able crops such as sugarcane into value added products comprising of refined sugar. It has the manufacturing facility in Jhang District. An increase in alternative uses of bagasse (i.e. cane wood, Bio fuel, CO<sub>2</sub> & Steel plant), has developed an interest in energy efficiency of which the boiler efficiency forms an essential part. The boiler efficiency not only depends on the boiler configuration and operation but also on the fuel being used. This paper describes a conventional sugar factory boiler, the analysis of boiler operation and describes the modifications applied for improving boiler efficiency.

Shakarganj has a capacity of 12000 TCD. Plant requires 263tph steam which includes 18tph steam required for the distillery. Sugarcane bagasse is used as boiler fuel containing 51-52% moisture. Brief introduction of boilers is as under:

### *Observations of Existing Boilers:*

The efficiency of the boiler at 75 tons/hr. (01 No. Boiler) was approx. 64-66% @ NCV. Estimated losses due to: Unburnt 10 to 12 % Radiation and convection losses: 6 to 8% Heat losses through stack: 18 to 20 % High combustion instability was observed in boiler through furnace ash analysis i.e. Presence of unburnt bagasse particles in

ash. High pressure losses at Ash Cyclones (16 mbar). Poor Air to Fuel ratio.

## REVIEW AND ANALYSIS OF BOILER OPERATION

**Major areas under analysis:** Boiler Configuration and Operation, Combustion stability, Boiler efficiency Comprehensive evaluation of boilers was done considering the above mentioned areas with the help of Flue gas analyzer. Below table shows some results of evaluation:

## INSTALLATIONS/ MODIFICATIONS APPLIED TO EXISTING BOILERS

Based on the outcomes of evaluation, modifications in the existing boilers were

applied. Modification and their results are discussed below.

### **Installation of Boiler retrofits:**

Installation of Re Grit Re-Firing system  
Installation of Pulsating Dampers  
Installation of Over firing nozzles  
Installation of Air distribution plate under the Grate  
Installation of Bio Gas burners  
Installation of Economizer

### **Modifications Applied:**

Modification of Ash Cyclones

### **RE GRIT RE FIRING SYSTEM:**

This system recovers the high unburnt bagasse from boiler through Ash hoppers and supplied with an Air duct from Secondary fan to push it in boiler furnace to use it again as fuel resulting in reduction in unburnt losses and saving bagasse.

### **Before System Installation:**

Fly ash was blackish that means high unburnt bagasse in boiler. (Unburnt losses were 10%-12%)

### **After System Installation Benefits:**

This System is used to recover unburnt bagasse from boiler. Unburnt bagasse is again fired in furnace to burn. Losses due to un burnt fuel reduced to 4 to 6% and increasing boiler efficiency by 2%.



**Re Grit Re Firing System installed at Boiler**

### **Bagasse Saving By Re Grit Re Firing System:**

Efficiency rise at average of 75 Ton/hr load	= 02%
Steam Saving at 01 Boiler	= $75 \times 0.02$ = 1.5 Tons/hr
Bagasse Saving at 01 Boiler	= $1.5 / 1.9$ = 0.789 Tons/hr
Bagasse Saving at 02 Nos. Boilers	= $0.789 \times 2$ = 1.57 Tons/Hr
Bagasse Saving per day	= $24 \times 1.57$ = <b>37.89 Tons/day</b>
Total amount saved per day	= $3000 \times 37.89$ <b>Rs 113,670/-</b>

### **BAGASSE SPREADING AND FIRING SYSTEM:**

Previously Bagasse spreading system was not so much efficient for feeding and spreading of bagasse over the full grate. Few improvements were done to make the existing system more efficient as below:

Improving the condition of the existing spreaders by proper

adjustment facilities of guide plate from outside. Installation of pulsating dampers with common drive so that the bagasse will be more equalized spread over the depth of the furnace. Installation of over firing nozzles in side walls of furnace and fed with the air from secondary air fans. Installation of air baffle in

hopper under the grate. Installation of twin roller feeders with picker roller in order to ensure continuous supply of bagasse and breaking of lumps so that fine particles can be spread.

### **PULSATING DAMPERS:**

Shakarganj has installed 02 types of Pulsating dampers Internal Pulsating Dampers

**Internal Pulsating Dampers****External Pulsating Dampers****External Pulsating Dampers****BIO GAS BURNER:**

As per previous practice biogas was blown into the boiler furnace without proper mixing it with combustion air. This resulted in in complete firing in boiler. Also resulting in high fouling and rapid corrosion of super heater.

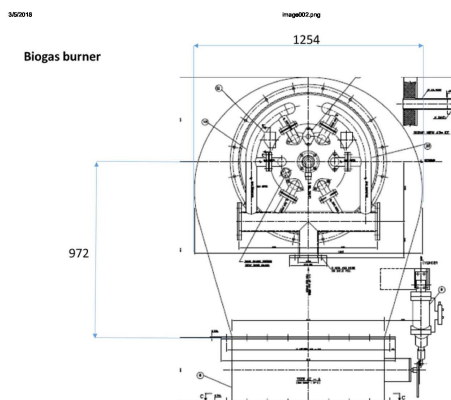
**Bio Gas Burner at Shakarganj:**

Biogas burners (Wes man Design) with air supply through existing FD Fan have been installed.

Using this burner made combustion and temperature stable, also air to fuel ratio improved. Uniform spreading of Bio gas with boiler

tubes. Which burner in furnace, hence minimizing it's direct contact with boiler tubes. Which decrease corrosion of tubes.

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**Bio Gas Burner Design****ECONOMIZER for Pre Heating of Boiler Feed Water:**

Generally economizers are used to reduce energy consumption. In Boiler their function is to pre heat the boiler feed water temperature. Captures the waste heat from

boiler flue gases and transfer it to the boiler feed water to raise the temperature up to 30°C.

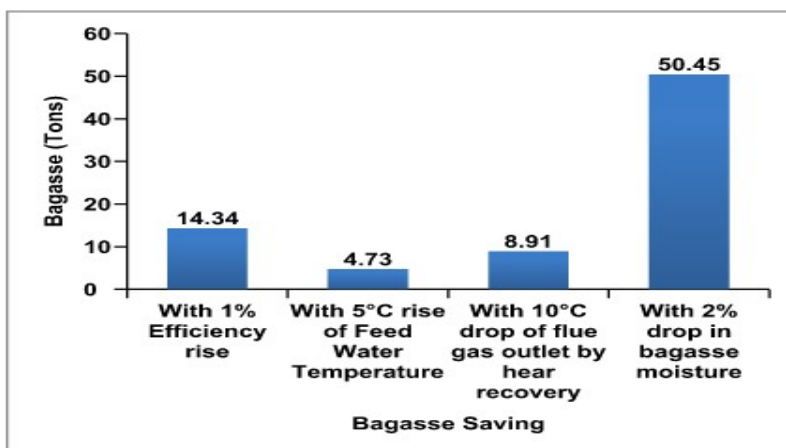
**Economizer Installed at Shakarganj Boilers:**

Shakarganj has installed economizers at 04 Nos. of

boilers with heating surface of 320 m<sup>2</sup>. It raises the feed water temperature about 30°C from 100-105°C to 130-135°C. Below is some calculated data for savings made by economizers.

Efficiency Rise With 10 °C temperature rise in feed water		= 01%
Efficiency with 20°C temperature rise of feed water. (01 Boiler)		= 02%
Steam Saving at One Boiler		= 75x0.02
Steam Saving at One Boiler		= 1.5 Tons/hr
Bagasse Saving at One Boiler	= 1.5/1.9	= 0.79 Tons/hr
Bagasse Saving at 04 Nos. Boiler	= 0.79x4	= 3.16 Tons/hr
Bagasse Saving per day	= 24x3.16	= <b>75.44 Tons/day</b>
Amount Saving	= 75.44x3000	<b>Rs. 227,520</b>

Below graph explicit the bagasse saving when compared with different parameters after evaluation.



**Fig.1 Bagasse saving vs Improved Boiler Parameters**

#### **THE ASH CYCLONES:**

Smaller cyclones replaced with greater size (5-6 % of existing cyclones size) to allow a pressure drop of 8 mbar at 80 tph capacity.

#### **Problems Faced:**

Ash Cyclone size was too small. Boiler was running overpressure at higher loads which made difficult to keep the boiler load in control. The angle of the swirls in cyclones was very small, which made the cyclone efficiency very low.

#### **Modifications:**

Size of cyclones increased (5-6 %) so that boiler can be run properly under pressure in furnace with maximum clearance. The angle of swirls also changed to improve the

dust collection. Pressure drop of 8 mbar at 80 tph capacity.

#### **CAPACITY ENHANCEMENT OF I.D FANS AT BOILER**

Volumetric capacity of all ID fans enhanced to 5600 m<sup>3</sup>/min from 4750 m<sup>3</sup>/min.

#### **BOILER EFFICIENCY**

Boiler efficiency is the percentage of heat input utilized in generation of steam. Efficiency of solid fuel boiler depends upon different factors like type of fuel, proper Where of fuel and combustion method. The easiest and most cost effective method is to calculate the efficiency value on five broad elements: Boiler stacks temperature,

Heat content of fuel, Fuel specification, Excess air levels & ambient air temperature and relative humidity.

**Boiler Efficiency = Energy  
Steam – Energy BFW x100**

#### **Energy Fuel**

This method is known as direct method which is based on simply that efficiency is equal to output divide. Where by input. The other method is the indirect method of calculating boiler efficiency. To account the boiler losses a better and precise formula for efficiency calculation is given below:

$$\eta = \frac{M_v}{NCV} \times 100$$



Where

Mv= Heat transfer to steam  
per kg of bagasse burnt  
(Kcal/Kg)

NCV=Net calorific value  
(Kcal/Kg)

$Mv = (N.C.V - Q) \cdot \alpha \cdot \beta \cdot \gamma$

Where

Q= Sensible heat loss in flue  
gas (Kcal/Kg)

$\alpha \cdot \beta \cdot \gamma$  = Co-efficient if x-tics of  
combustion efficiency

$\alpha$  = Co-efficient representing  
heat loss due to un-burnt  
2solids.

B= Co-efficient to account for  
heat losses by radiation.

$\gamma$  = Co-efficient of incomplete  
combustion.

For spreader stoker  
furnaces, its normal value  $\alpha$   
is taken as 0.975.  $\beta$  value  
varies from 0.95 to 0.99 for  
more or less efficient  
lagging.  $\gamma$  value is taken as  
0.97.

$Q = [(1-M) \times (1.4RA - 0.13) + 0.5] t$

Where:

M= % Moisture in Bagasse

RA=Ratio of excess Air

t=Temperature of Flue  
gases

Here RA is ratio of excess  
air usually taken 1.45 for  
bagasse.

#### Calorific Value:

There are two different  
calorific values, a gross  
calorific value (GCV) and a  
net calo calorific rific value  
(NCV).

The GCV is the total energy  
released during the  
combustion process and can  
only be accurately determined  
by using a bomb calorimeter.

The NCV is the GCV minus  
the latent heat of the water  
formed by the combustion  
process and is obtained by  
calculation. The experimental  
procedure and method of  
calculation are laid down in  
ISO 1928 (Anon 1995).

Some comparison of bagasse  
NCV vs Moisture is shown in  
below graph:

The Boiler efficiency is also  
linked with flue gas  
temperature at the outlet.  
Trend with comparison is  
shown as below.

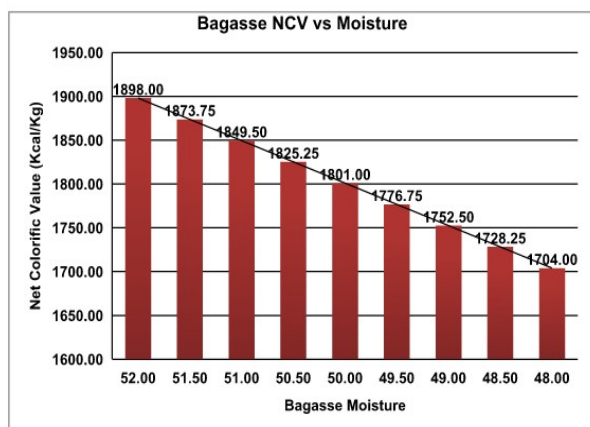


Fig.2 Bagasse NCV vs Moisture

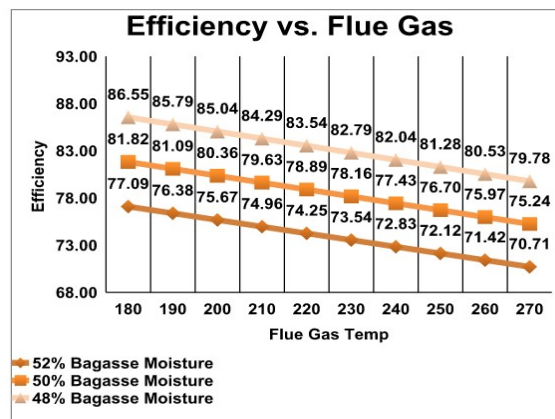


Fig.3 Boiler Efficiency vs Flue Gas Temp.

## CONCLUSION

The modifications done in the  
existing boilers system has  
given much improvement and  
benefits for performance  
optimization and efficiency

enhancement of existing  
boilers. i.e.

#### Boiler Efficiency:

Increase from 66% to 76%

Re Grit Re-Firing System:

Increased efficiency by 2%

Bagasse

Saving 37.89 Tons/Day  
Un burnt losses reduced  
4-6%

#### Pulsating dampers:

Uniform spreading of  
bagasse

**Bio Gas Burner:**

Complete Combustion of bio gas, Minimized corrosion formation on tubes

**Economizer:**

20°C Feed Water Temperature raise  
02% efficiency rise  
Bagasse Saving 75.44 Tons/Day  
Increase in size of Ash

**Cyclones:**

Boiler runs under pressure

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**Table-1 Boilers Installed at Shakarganj**

Design	Type	Heating Surface (m <sup>2</sup> )	Steam Production (Tons/Hr.)	Type of Fuel	Operating Pressure Kg / cm <sup>2</sup>	Operating Temperature
Babcock & Wilcox	Water Tube Boiler	1,361	40	Bagasse/ Bio Gas/ Sui Gas/ Furnace Oil	25	350 °C
FCB Franc		2,071	80			
Yoshimini Japan		2,220	80			
Yoshimini Japan		2,220	80			
Yoshimini Japan		2,220	80 at NCR			

**Table-2 Evaluation Results with Flue Gas Analyzer**

Description	Value	Description	Value
Furnace Pressure	-4.38 mmH <sub>2</sub> O	Flue Gas Outlet Temperature Boiler	321 °C
Flue Gas Temperature Outlet Flue Gas Arrestor	187 °C	Air Temperature Air to Grate	173 °C
O <sub>2</sub> Flue Gas Outlet Boiler	2.8/3 Vol%dry	CO Flue Gas Outlet Boiler	2500 ppmv
O <sub>2</sub> Flue Outlet ID Fan	5.8-6.8 Vol%dry	CO Outlet ID Fan	2000-8000 ppmv
Flue Gas Pressure at ID Fan suction	-2 mbar	Flue Gas Pressure at Outlet ID Fan	-18 mbar
Air Pressure at Outlet FD Fan	7 mbar	Air Pressure at Outlet Secondary Air Fan	20 mbar
Pressure Drop Over Fly Ash Arrestor	16 mbar	Calculated Pressure Drop Over Fly Ash Arrestor	16.4 mbar

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